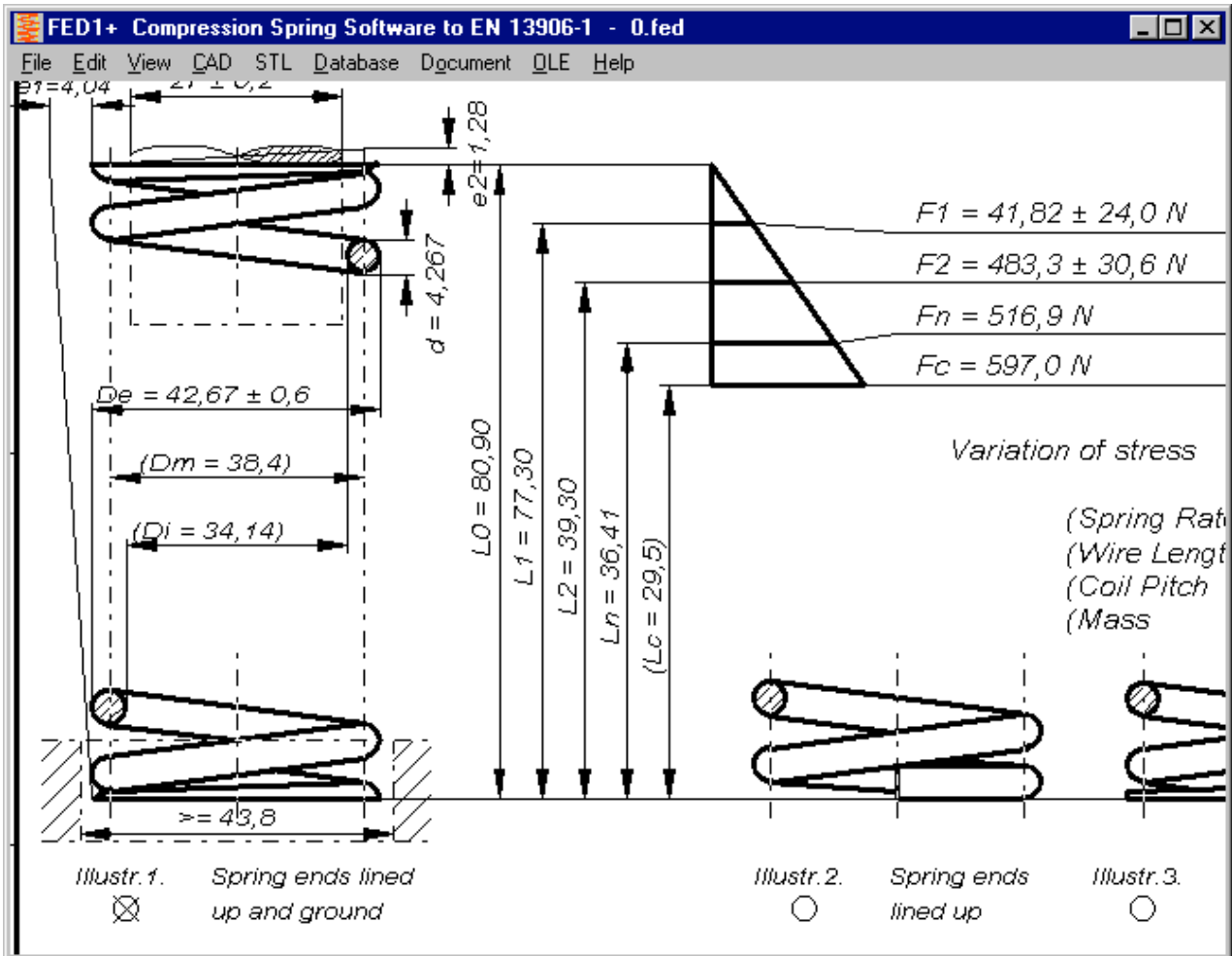
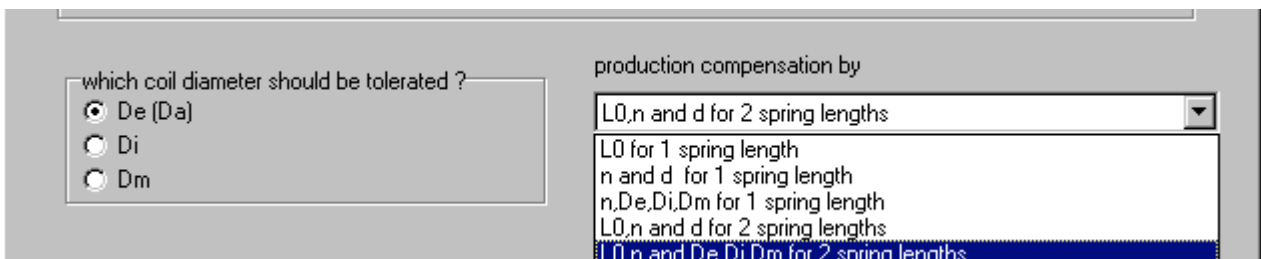


by Fritz Ruoss

**FED1+: (Lc = x.x) in Production Drawing**



If production compensation is defined by number of coils (n), block length cannot be a control dimension. Block length Lc is printed without tolerance in this case. Additionally, it is drawn in brackets in the latest version of FED1+.



### FED15, FED16: Input Material Data

As alternative to pick from database, you can input base properties of material (E module, tensile strength, yield strength, permissible bending stress) directly now.

20: EN 10089 52CrMoV4 hot-rolled spring steel 1.7701 Database

Select

Database fedwst.dbf (priority)

Database fedwst.dbf (complete)

others .....

tolerance t, W[b]

EN 10140-A (t<= 10mm, W<600mm)

t = 36 ± 0,090 mm

W = 80 ± 0,500 mm

material

material EN 10089 52CrMoV4

modulus of elasticity 206000 MPa

density 7,85 kg/dm<sup>3</sup>

tensile strength 1450 MPa

adm. bending stress 1088 MPa

surface

drawn

rolled

ground

shaved

OK Cancel Help

### FED3+: Tolerance of bending angle corrected

```
clamped leg: fixed clamped, bent-up, int.
-----
bending radius          r          mm          2 + 1,00
tolerance bending angle A phi  Aphi1    °          ± 6,0
leg length              L          mm          50 ± 1,48
-----
moved leg: fixed clamped, tangential
leg length              L          mm          50 ± 1,48
```

Tolerance of bending angle according to quality 1,2,3 of DIN 2194 of torsion springs with bent legs was not calculated correctly. Tolerance Aphi is listed in the standard printout only, not in production drawing nor Quick view. Thanks to Mr. Demmelbauer of Hutter & Schrantz steel springs for hint and documents.

### ZAR1+ .. ZAR8: Modifications in Material database

Properties were slightly modified for nodular cast iron:

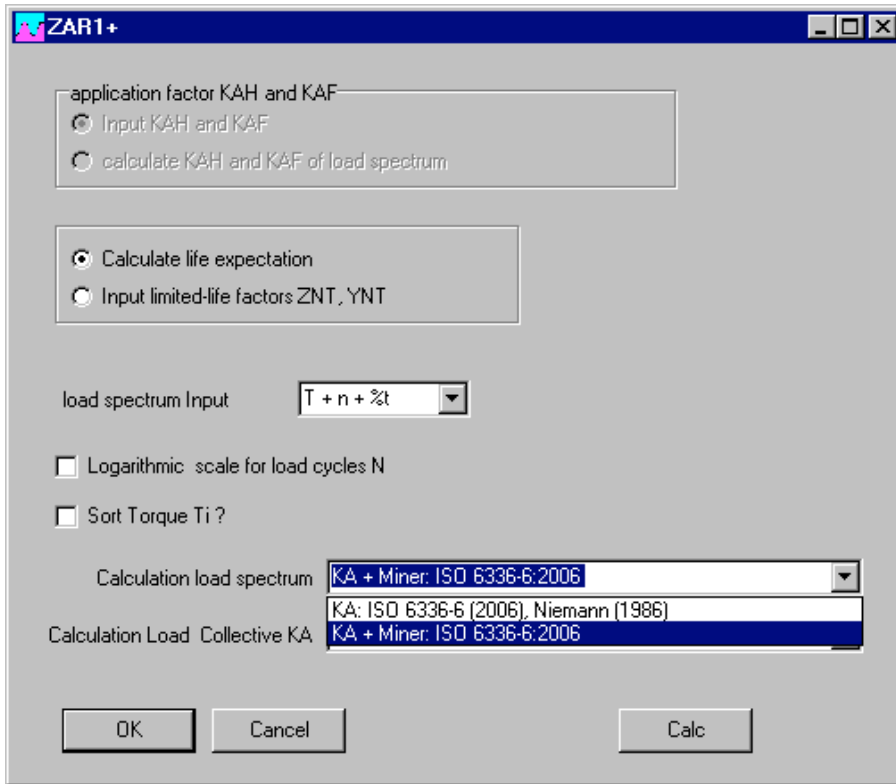
EN-GJS-400: HB from 180 into 170

EN-GJS-600: E-Module from 180 GPa into 174 Gpa

EN-GJS-1000: E-Module from 190 GPa into 168 GPa,  $\mu$  from 0.29 into 0.27, density from 7.2 into 7.1.

Thanks to Mr. Schulze of BS Technik for hints and documents.

**ZAR1+,ZAR2,ZAR5,ZAR6,ZAR7,ZAR8: Load spectrum according to ISO 6336-6**



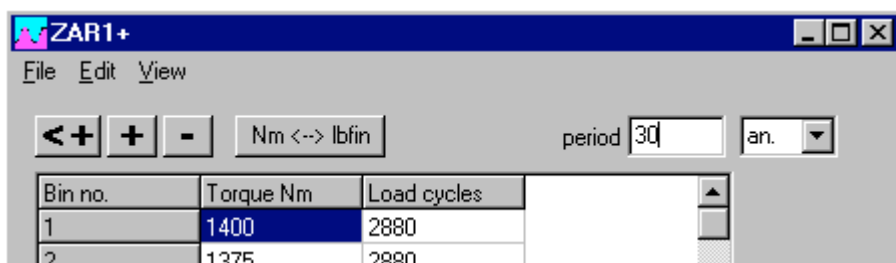
Until now, our ZAR-Software used calculation method according to Niemann to calculate application factors KA from load spectrum. This method is described in ISO 6336-6 annex A with purpose of a first estimation only. So we have added the preferred method to Palmgren-Miner as described in ISO 6336-6. Select both methods at Edit\Load spectrum\Config.

Calculation must be separate for gear 1 and gear 2 to this method. Gear 2 differs in speed and number of cycles by transmission ratio and number of tooth contacts per rotation:

$$N2 = N1 * z1/z2 * e2/e1$$

with 1 = pinion, 2 = wheel, N = number of cycles, z = number of teeth, e = tooth contacts per rotation

**ZAR1+,ZAR2,ZAR5,ZAR6,ZAR7,ZAR8: Input Period**



New input is period time for the load spectrum (including idle time). Idle time is calculated as difference of period and sum of load spectrum. If input is 0 or less than sum of load spectrum, idle time is 0 and period is set to sum of load spectrum. Normally, required lifetime of the gear is set as period time. But you can also describe the load spectrum for a period of days or hours. Period time unit can be set to seconds, minutes, hours, days or years.

### ZAR1+,ZAR2,ZAR5,ZAR6,ZAR7,ZAR8: Load spectrum: lifetime to ISO 6336-6

Both calculation methods (KA and Miner) can be compared by comparing the calculated lifetime. Because idle time can be considered now in Miner calculation, but not in KA calculation, one must compare lifetime at 100% duty cycle.

Calculation of lifetime by Miner sum considers all load bins, KA calculation considers only bins with torque higher than 50% of nominal torque or until the sum of cycles is higher than fatigue safe limit of the S/N curve.

Application example table 2 of ISO 6336-6 calculates 3900 hours until pitting for the pinion to KA method, and 6000 hours according to Miner method. If idle time of 6E6 s in 70 days is considered, time until pitting (gear1) is 5455621 hours = 630 years. ISO 6336 annex C calculates 30 years with a safety factor of 1.428. Idle time of 99,9% is extremely high in this example, in 30 years the gear is on duty for 10 days only.

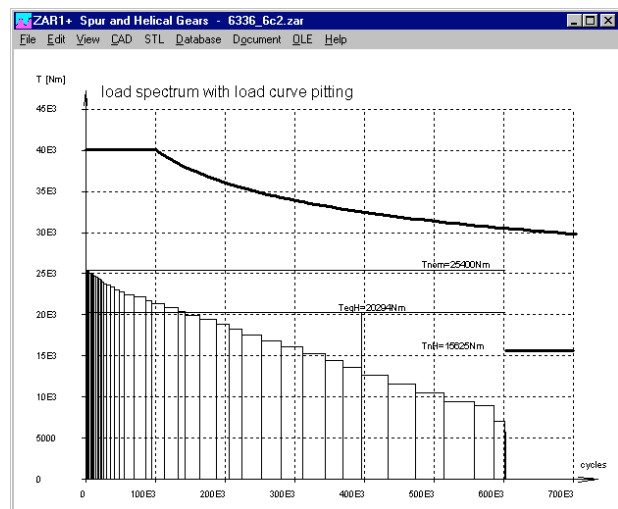
load spectrum		Case carburized	
N sum	3,83E3	N0F	1E3
N eqF	2,52E3	NnF	3E6
N eqH	2,52E3	p F	8,7
KA F	0,81	NDH	1E5
KA H	0,80	NnH	5E7
		p H	6,6

load spectrum	
T N	Nm 25400
T eqF	Nm 20693
T eqH	Nm 20294
T nF	Nm 28517
T nH	Nm 15625
T 0F	Nm 71577
T 0H	Nm 40065

KA, SH, Nf, tF	gear 1	gear 2
KAH	0,80	0,80
SH	0,87	0,87
Cycles to Failure NfH	8,2E6	8,2E6
Time to fail. tFH 100% h	3898	13759
KAF	0,81	0,81
SF	1,38	1,44
Cycles to Failure NfF	>1,0E10	>1,0E10



### ZAR1+,ZAR2,ZAR5,ZAR6,ZAR7,ZAR8: Load spectrum diagrams

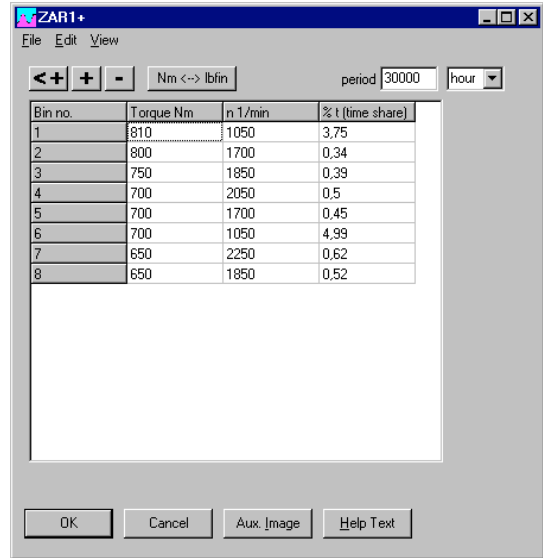
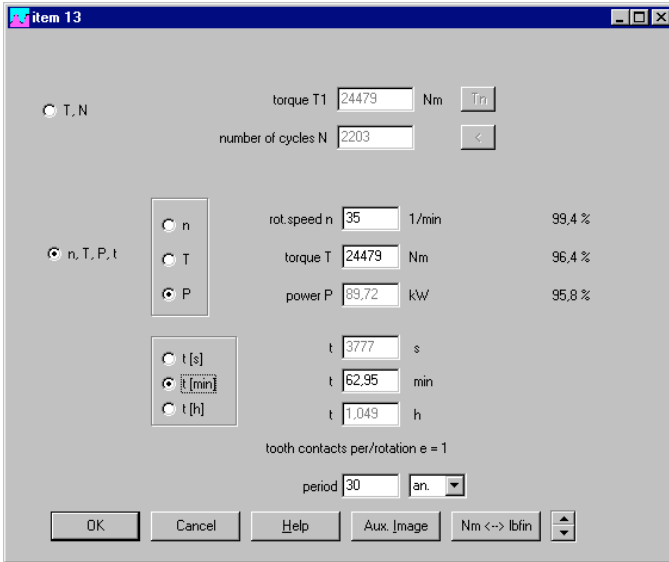
Limits of the y axis (torque) has been changed, so that S/N curve can be drawn in any case. Equivalent S/N curve of the load spectrum is no longer drawn nor the corresponding parameters TnFeq, TnHeq, T0Feq and T0Heq, as not mentioned in ISO 6336-6. Scale of the x axis (cycles) can be drawn linear or logarithmic. Program calculates torque for safety factors SH=1 and SF=1 in an iteration, as well as static safety SHplast and SFplast for the highest (1<sup>st</sup>) torque of the load spectrum.

```

ZAR1+  c:\temp\outwin.txt
File Edit
| Load spectrum of ISO 6336-6:2006 Table C.2 and C.3 |
| ZAR1+ calculates miner sum and periods to failure (30 years) |
+-----+
torque T1(SF=1, KA=1)          T1 SF1   Nm      28517
-----
torque T1(SH=1, KA=1)          T1 SH1   Nm      15625
-----
safety plast, T1=25423Nm        SFplast  Nm        2,80
-----
safety plast, T1=25423Nm        SHplast  Nm        1,24
-----

```

## ZAR1+,ZAR2,ZAR5,ZAR6,ZAR7,ZAR8: Input load bins with different speed

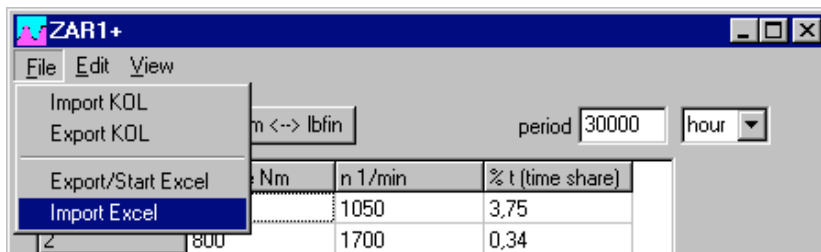


In standard input of load spectrum, you can input torque or power and speed, program calculates and shows relation to nominal values in %.

At input „n,T,P,t“ you enter speed, torque or power and time for each bin and program calculates cycles. These data have not been saved until now (T and N only), values of n,P,t cannot be recalculated if speed varies. Now, speed of each bin will be saved. If not saved, nominal speed is used for all bins.

Also at "Edit\Load Spectrum\Table" you can input speed for each bin.

## ZAR1+,ZAR2,ZAR5,ZAR6,ZAR7,ZAR8: Load spectrum Export/Import



For exchange with MS-Excel a new Export/Import Excel has been added in the menu. Or you can use Copy and Paste for import/export of load spectrum tables with any Windows spreadsheet. Furthermore, you can import or export load bins by means of KOL file. The proprietary KOL format was extended by speed for each bin.

## ZAR1+,ZAR2,ZAR5,ZAR6,ZAR7,ZAR8: Sort Load Spectrum



Until now, you had to input bins of load cycles in correct order, beginning with the highest torque. Now you can enter bins unsorted, and let sort by ZAR software.

## ZAR1+,ZAR2,ZAR5,ZAR6,ZAR7,ZAR8: Load spectrum tables

ZAR1+ Spur and Helical Gears - lastkbs.zar

i	T [Nm]	N	n rpm	i	SH2	ZNT2	N2	NfH2	U(N2/NfH)	N2 %	fH2 %
1	810	7,09E7	1050	1	0,83	1,20	2,0E7	4,5E6	4,330	26,9	51,1
2	800	1,04E7	1700	2	0,85	1,18	2,9E6	5,4E6	5,28E-1	4,0	6,2
3	750	1,30E7	1850	3	0,88	1,14	3,6E6	8,7E6	4,14E-1	4,9	4,9
4	700	1,85E7	2050	4	0,91	1,10	5,1E6	1,4E7	3,56E-1	7,0	4,2
5	700	1,38E7	1700	5	0,91	1,10	3,8E6	1,4E7	2,76E-1	5,2	3,3
6	700	9,43E7	1050	6	0,90	1,11	2,6E7	1,2E7	2,084	35,8	24,6
7	650	2,51E7	2250	7	0,95	1,06	6,9E6	2,4E7	2,84E-1	9,5	3,4
8	650	1,73E7	1850	8	0,94	1,06	4,8E6	2,3E7	2,04E-1	6,6	2,4
sum		2,63E8		sum			7,3E7	Miner	8,477	100,0	100,0

i	SF2	YNT2	N2	NfF2	U(N2/NfF)	N2 %	fF2 %
1	1,05	0,95	2,0E7	4,2E7	4,68E-1	26,9	89,8
2	1,06	0,94	2,9E6	6,2E7	4,65E-2	4,0	8,9
3	1,13	0,88	3,6E6	1,7E9	2,16E-3	4,9	0,4
4	1,22	0,82	5,1E6	1,0E10	5,09E-4	7,0	0,1
5	1,22	0,82	3,8E6	1,0E10	3,80E-4	5,2	0,1
6	1,23	0,81	2,6E7	1,0E10	2,60E-3	35,8	0,5
7	1,31	0,76	6,9E6	1,0E10	6,93E-4	9,5	0,1
8	1,32	0,76	4,8E6	1,0E10	4,78E-4	6,6	0,1
sum			7,3E7	Miner	5,21E-1	100,0	100,0

Tables with intermediate results of load spectrum calculation to Palmgren-Miner can be listed in tables. Share of cycles (N %) and share of damage (tF %) is listed for each bin (similar as in FED1+ load spectrum). Share of damage for each bin is calculated as  $f_i = U_i / \text{Miner sum}$ . Cause 4 Miner sums have to be calculated (pitting, tooth root stress gear 1 and gear 2), you get a huge number of data that can be listed in 3 tables.

ZAR1+ Spur and Helical Gears - lastkbs.zar

i	T [Nm]	N	n rpm	i	SH1	SH2	ZNT1	ZNT2	NfH1	NfH2	U(N1/NfH)	U(N2/NfH)	i	SF1	SF2	YNT1	YNT2	NfF1	NfF2	U(N1/NfF)	U(N2/NfF)
1	810	7,09E7	1050	1	0,83	0,83	1,20	1,20	4,5E6	4,5E6	1,57E1	4,330	1	1,14	1,05	0,88	0,95	1,8E9	4,2E7	3,93E-2	4,68E-1
2	800	1,04E7	1700	2	0,85	0,85	1,18	1,18	5,4E6	5,4E6	1,915	5,28E-1	2	1,15	1,06	0,87	0,94	2,7E9	6,2E7	3,91E-3	4,65E-2
3	750	1,30E7	1850	3	0,88	0,88	1,14	1,14	8,7E6	8,7E6	1,5	4,14E-1	3	1,22	1,13	0,82	0,88	1,0E10	1,7E9	1,30E-3	2,16E-3
4	700	1,85E7	2050	4	0,91	0,91	1,10	1,10	1,4E7	1,4E7	1,292	3,56E-1	4	1,31	1,22	0,76	0,82	1,0E10	1,0E10	1,85E-3	5,09E-4
5	700	1,38E7	1700	5	0,91	0,91	1,10	1,10	1,4E7	1,4E7	1,002	2,76E-1	5	1,32	1,22	0,76	0,82	1,0E10	1,0E10	1,38E-3	3,80E-4
6	700	9,43E7	1050	6	0,90	0,90	1,11	1,11	1,2E7	1,2E7	7,554	2,084	6	1,32	1,23	0,76	0,81	1,0E10	1,0E10	9,43E-3	2,60E-3
7	650	2,51E7	2250	7	0,95	0,95	1,06	1,06	2,4E7	2,4E7	1,030	2,84E-1	7	1,42	1,31	0,71	0,76	1,0E10	1,0E10	2,51E-3	6,93E-4
8	650	1,73E7	1850	8	0,94	0,94	1,06	1,06	2,3E7	2,3E7	7,38E-1	2,04E-1	8	1,42	1,32	0,70	0,76	1,0E10	1,0E10	1,73E-3	4,78E-4
sum		2,63E8		sum					Miner	3,07E1	8,477		sum					Miner	6,14E-2	5,21E-1	

## SR1+: Self-defined elasticity for calculation of deltaP

thickness L  mm

database clamp.plate (Washer)

input Elasticity ?

Elasticity delta  mm/N

material

pG  MPa

Re (Rp0.2)  MPa

alpha T  mm/K

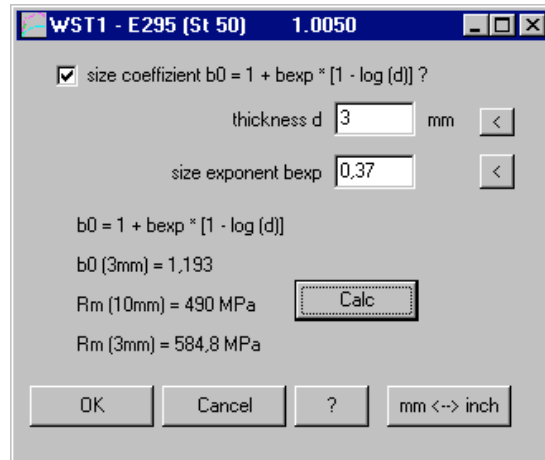
Young's modulus  MPa

pGKr  MPa

Tmax  °C

At input of clamping plates, you can input elasticity as alternative to calculation by SR1+. Now it was found that self-defined elasticity was considered only if "deformation sleeve" was configured. New version now considers self-defined elasticity for both, calculation by deformation sleeves and by deformation cone. Thanks to Mr. Huegl of Leadec Engineering for hint and documents.

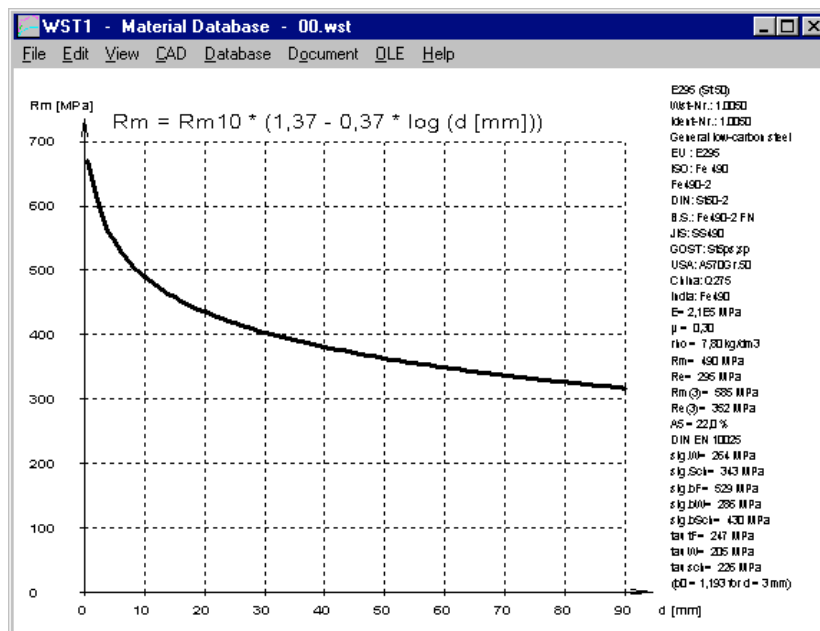
## WST1: Calculate size factor from size exponent and material thickness



Until now, WST1 calculated size factor for material groups 1 to 27 (steel) to the formula  $b_0=1+0.2*[1-\log(d)]$ . Material properties of the database are based on material thickness (material diameter) of 10 mm. Cause size factor for different materials and different heat treatment differs, now you can input material exponent for calculation of the size factor. First you have to hook on that size factor must be considered. Material exponent for steel lies between 0.1 and 0.4.

$$\sigma(d) = \sigma(d_{10}) * b_0(d)$$

$$b_0(d) = 1 + b_{exp} [1 - \log (d)]$$



## WST1: Fatigue strength 1.4310 and 1.4568

Fatigue strength stress limits for 1.4310 and 1.4568 have been added, so that Smith-Diagram, Haigh-Diagram, Goodman-Diagram and S/N curve can be drawn for these materials.

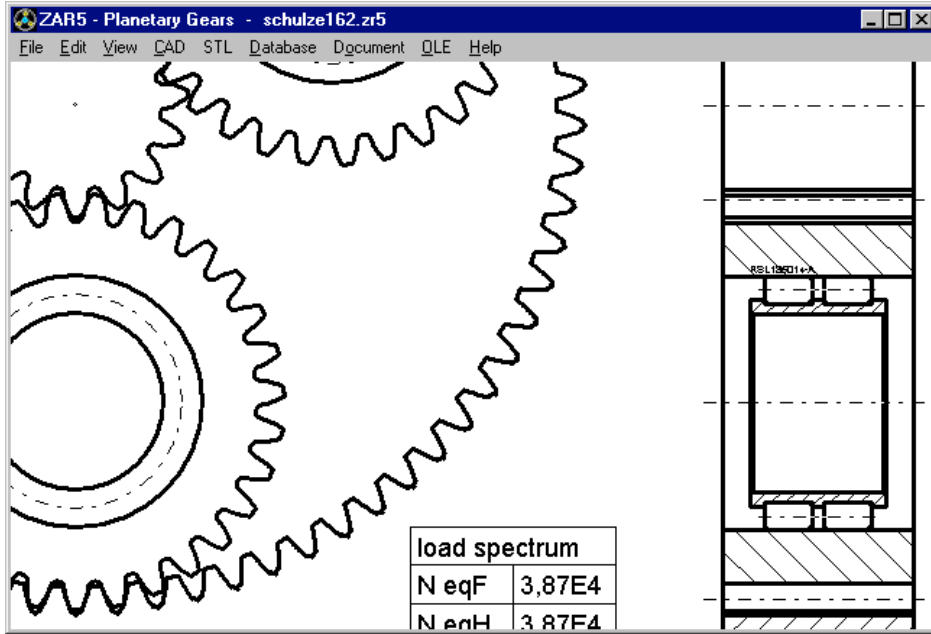
## WST1: Nodular cast iron: new materials and modifications

Modifications of E module and stress limits for this materials:

EN-GJS-400, EN-GJS-500, EN-GJS-700, EN-GJS-800

New materials in WST1: EN-GJS-1000-5, EN-GJS-1050-6, EN-GJS-1200-2, EN-GJS-1400-1, EN-GJV-400

**LG1, ZAR5, ZAR7, ZAR8, WL1+: Full cplm. needle bearing and roller bearing**



If not enough space for roller bearings with inner and outer ring, bearing of planet wheels can be a cylindrical roller bearing without outer ring, or needles only between planet bolt and planet wheel bore without cage.

The screenshot shows a dialog box titled 'bearing load Fr = 62793 N Fa = 0 N ZAR5 database Full cplm.cyl.roller bearing'. It contains a table with the following data:

NAME	DI	DEW	B	C	C0	CU	M	D11
RSL185014-A	70	100,28	54	233000	350000	0	1,12	
RSL182214-A	70	111,01	31	181000	223000	0	0,98	
RSL183015-A	75	107,9	30	162000	194000	0	0,73	
RSL185015-A	75	107,9	54	245000	385000	0	1,46	
RSL182215-A	75	115,78	31	187000	236000	0	1,03	
RSL183016-A	80	116,99	34	173000	224000	0	0,97	

For full compl. needle bearings, clearance between rollers is calculated by the program (pitch diameter clearance  $TES = 5E-3 Z$  according to Schaeffler). Then roller race diameter of shaft and bore can be calculated. To select full compl.needle bearings as known from other roller bearing databases, a new database file was created for pin diameters of 1mm until 6mm and pin length of 5.8mm until 39.8mm and 10 until 50 pins, with calculated race diameters, rated load C and C0 and mass. This results in a new database file with 2500 records.

The screenshot shows a dialog box titled 'bearing load Fr = 62793 N Fa = 0 N ZAR5 database Full compl.needle bearing'. It contains a table with the following data:

NAME	DFW	DEW	B	Dw	Z	C	C0	M
NRB3,5x29,8-Z44	45,631	52,631	29,8	3,5	44	70564	183764	99,029
NRB3,5x34,8-Z44	45,631	52,631	34,8	3,5	44	80018	215231	115,645
NRB6x17,8-Z27	45,731	57,731	17,8	6	27	61764	108382	106,671
NRB4x11,8-Z39	45,772	53,772	11,8	4	39	35001	70699	45,397
NRB4x13,8-Z39	45,772	53,772	13,8	4	39	40085	83323	53,091
NRB4x15,8-Z39	45,772	53,772	15,8	4	39	44989	95948	60,786
NRB4x17,8-Z39	45,772	53,772	17,8	4	39	49746	108573	68,48
NRB4x19,8-Z39	45,772	53,772	19,8	4	39	54375	121197	76,174
NRB4x21,8-Z39	45,772	53,772	21,8	4	39	58895	133822	83,869
NRB4x23,8-Z39	45,772	53,772	23,8	4	39	63317	146447	91,563
NRB4x25,8-Z39	45,772	53,772	25,8	4	39	67653	159072	99,258
NRB4x27,8-Z39	45,772	53,772	27,8	4	39	71911	171696	106,952
NRB4x29,8-Z39	45,772	53,772	29,8	4	39	76098	184321	114,646



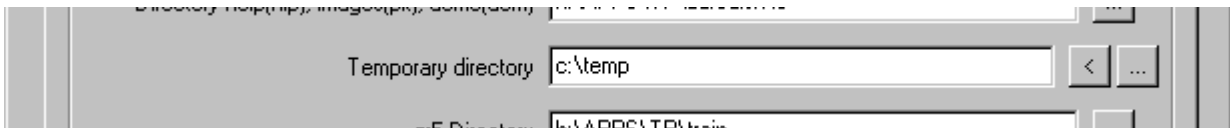
## Windows 10 – Automatic Updates

Some customers got a key code error after Windows 10 automatic update. The reason is, that sometimes Windows 10 update modifies partitions of the hard disk, so that after update the size of the hard disk shrinks by 0.5 GB. I do not know what Microsoft makes with the hidden partition. Anyway, you have to delete \*.cod files in the HEXAGON program folders, then run programs and send key code requests to get new key codes.

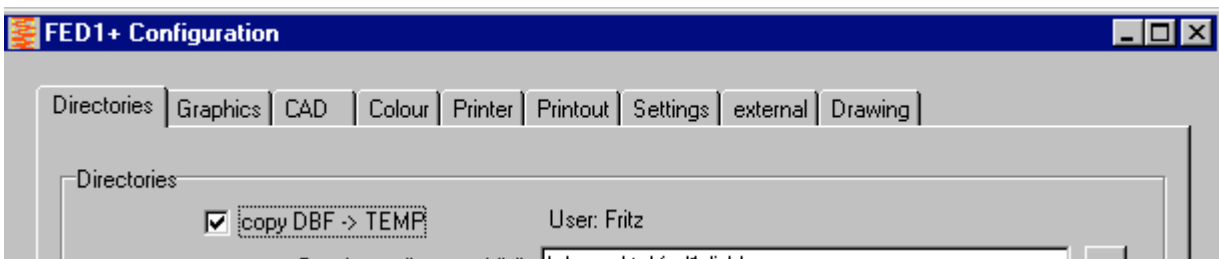
### Tip: Where to install HEXAGON-Software

If your computer has more than one hard disks or logical drives, better install HEXAGON software on another drive besides Windows system drive, if your operating system is Windows 10.

If your PC has a small and fast SSD and a large, slower hard disk, install HEXAGON software on the (slow) hard disk, if Windows 10 runs on the SSD. If you want to accelerate HEXAGON software by means of your SSD, then configure a folder on the SSD as temporary drive (at "File\Settings\Directories").



### Tip: Network version: Set "copy DBF -> TEMP"



This new option copies all dbf files at program start into the temp folder. This accelerates program speed enormously and reduces network traffic, if temp directory is a local drive.

Temporary drive must be as fast as possible (RAM-Disk, SSD or hard disk, but no network drive!)

### Tip: Program start by double click into calculation file reports error after update?

HEXAGON software reports a database error since update or change of program path? Probably cfg file with configured directories could not be found. Copy cfg file into "c:\hexagon\", then it should start without error.

**HEXAGON PRICELIST 2017-07-01**

<b>PRODUCT</b>	<b>EUR</b>
DI1 Version 1.2 O-Ring Seal Software	190,-
DXF-Manager Version 9.0	383,-
DXFPLOT V 3.2	123,-
FED1+ V29.6 Helical Compression Springs incl. spring database, animation, relax., 3D,..	695,-
FED2+ V20.2 Helical Extension Springs incl. spring database, animation, relaxation, ...	675,-
FED3+ V19.0 Helical Torsion Springs incl. prod.drawing, animation, 3D, rectang.wire, ...	480,-
FED4 Version 7.3 Disk Springs	430,-
FED5 Version 15.7 Conical Compression Springs	741,-
FED6 Version 16.3 Nonlinear Cylindrical Compression Springs	634,-
FED7 Version 13.2 Nonlinear Compression Springs	660,-
FED8 Version 6.9 Torsion Bar	317,-
FED9 Version 6.0 Spiral Spring	394,-
FED10 Version 3.5 Leaf Spring (complex)	500,-
FED11 Version 3.3 Spring Lock and Bushing	210,-
FED12 Version 2.4 Elastomere Compression Spring	220,-
FED13 Version 4.0 Wave Spring Washers	228,-
FED14 Version 1.4 Helical Wave Spring	395,-
FED15 Version 1.4 Leaf Spring (simple)	180,-
FED16 Version 1.1 Constant Force Spring	225,-
FED17 Version 1.0 Magazine Spring	725,-
GEO1+ V6.1 Cross Section Calculation incl. profile database	294,-
GEO2 V2.6 Rotation Bodies	194,-
GEO3 V3.3 Hertzian Pressure	205,-
GEO4 V4.2 Cam Software	265,-
GEO5 V1.0 Geneva Drive Mechanism Software	218,-
GR1 V2.0 Gear construction kit software	185,-
HPGL-Manager Version 9.0	383,-
LG1 V6.4 Roll-Contact Bearings	296,-
LG2 V2.2 Hydrodynamic Plain Journal Bearings	460,-
SR1 V22.3 Bolted Joint Design	640,-
SR1+ V22.3 Bolted Joint Design incl. Flange calculation	750,-
TOL1 V11.8 Tolerance Analysis	506,-
TOL1CON V1.5 Conversion Program for TOL1	281,-
TOL2 Version 3.3 Tolerance Analysis	495,-
TOLPASS V4.1 Library for ISO tolerances	107,-
TR1 V4.0 Girder Calculation	757,-
WL1+ V20.1 Shaft Calculation incl. Roll-contact Bearings	945,-
WN1 Version 11.6 Cylindrical and Conical Press Fits	485,-
WN2 V10.0 Involute Splines to DIN 5480	250,-
WN2+ V10.0 Involute Splines to DIN 5480 and non-standard involute splines	380,-
WN3 V 5.4 Parallel Key Joints to DIN 6885, ANSI B17.1, DIN 6892	245,-
WN4 V 4.6 Involute Splines to ANSI B 92.1	276,-
WN5 V 4.6 Involute Splines to ISO 4156 and ANSI B 92.2 M	255,-
WN6 V 3.0 Polygon Profiles P3G to DIN 32711	180,-
WN7 V 3.0 Polygon Profiles P4C to DIN 32712	175,-
WN8 V 2.2 Serration to DIN 5481	195,-
WN9 V 2.2 Spline Shafts to DIN ISO 14	170,-
WN10 V 4.1 Involute Splines to DIN 5482	260,-
WN11 V 1.3 Woodruff Key Joints	240,-
WNXE V 2.0 Involute Splines - dimensions, graphic, measure	375,-
WNXK V 2.0 Serration Splines - dimensions, graphic, measure	230,-
WST1 V 10.02 Material Database	235,-
ZAR1+ V 26.0 Spur and Helical Gears	1115,-
ZAR2 V7.9 Spiral Bevel Gears to Klingelnberg	792,-
ZAR3+ V8.9 Cylindrical Worm Gears	620,-
ZAR4 V5.2 Non-circular Spur Gears	1610,-
ZAR5 V11.5 Planetary Gearings	1355,-
ZAR6 V3.9 Straight/Helical/Spiral Bevel Gears	585,-
ZAR7 V1.4 Plus Planetary Gears	1380,-

ZAR8 V1.4 Ravigneaux Planetary Gears	1950,-
ZARXP V2.1 Involute Profiles - dimensions, graphic, measure	275,-
ZAR1W V1.7 Gear Wheel Dimensions, tolerances, measure	450,-
ZM1.V2.5 Chain Gear Design	326,-

PACKAGES	EUR
<b>HEXAGON Mechanical Engineering Package</b> (TOL1, ZAR1+, ZAR2, ZAR3+, ZAR5, ZAR6, WL1+, WN1, WN2+, WN3, WST1, SR1+, FED1+, FED2+, FED3+, FED4, ZARXP, TOLPASS, LG1, DXFPLOT, GEO1+, TOL2, GEO2, GEO3, ZM1, WN6, WN7, LG2, FED12, FED13, WN8, WN9, WN11, DI1, FED15, WNXE, GR1)	8,500.-
<b>HEXAGON Mechanical Engineering Base Package</b> (ZAR1+, ZAR3+, ZAR5, ZAR6, WL1+, WN1, WST1, SR1+, FED1+, FED2+, FED3+)	4.900.-
<b>HEXAGON Spur Gear Package</b> (ZAR1+ and ZAR5)	1,585.-
<b>HEXAGON Planetary Gear Package</b> (ZAR1+, ZAR5, ZAR7, ZAR8, GR1)	3,600.-
<b>HEXAGON Involute Spline Package</b> (WN2+, WN4, WN5, WN10, WNXE)	1,200.-
<b>HEXAGON Graphic Package</b> (DXF-Manager, HPGL-Manager, DXFPLOT)	741.-
<b>HEXAGON Helical Spring Package</b> (FED1+, FED2+, FED3+, FED5, FED6, FED7)	2,550.-
<b>HEXAGON Tolerance Package</b> (TOL1, TOL1CON, TOL2, TOLPASS)	945.-
<b>HEXAGON Complete Package</b> (All Programs of Engineering Package, Graphics Package, Tolerance Package, Helical Spring Package, Planetary Gear Package, TR1, FED8, FED9, FED10, ZAR4, GEO4, WN4, WN5, FED11, WN10, ZAR1W, FED14, WNXK, FED16, FED17, GEO5)	12,900.-

#### Quantity Discount for Individual Licenses

Licenses	2	3	4	5	6	7	8	9	>9
Discount %	25%	27.5%	30%	32.5%	35%	37.5%	40%	42.5%	45%

#### Network Floating License

Licenses	1	2	3	4	5	6	7..8	9..11	>11
Discount/Add.cost	-50%	-20%	0%	10%	15%	20%	25%	30%	35%

(Negative Discount means additional cost)

#### Language Version:

- **German and English** : all Programs
- **French**: FED1+, FED2+, FED3+, FED4, FED5, FED6, FED7, FED9, FED10, FED13, FED14, FED15, TOL1, TOL2.
- **Italiano**: FED1+, FED2+, FED3+, FED4, FED5, FED6, FED7, FED9, FED13, FED14, FED17.
- **Swedish**: FED1+, FED2+, FED3+, FED5, FED6, FED7.
- **Portugues**: FED1+, FED17
- **Spanish**: FED1+, FED2+, FED3+, FED17

#### Updates:

Update prices	EUR
Software Update (software + pdf manual)	40,-
Software Update (software 64-bit Win + pdf manual)	50,-

Update Mechanical Engineering Package: 800 EUR, Update Complete Package: 1000 EUR

**Maintenance contract** for free updates: annual fee: 150 EUR + 40 EUR per program

#### Hexagon Software Network Licenses

Floating License in the time-sharing manner by integrated license manager

Individual licenses may not be installed in a network!

#### Conditions for delivery and payment

General packaging and postage costs are EUR 60, (EUR 25 inside Europe)

Delivery by Email (zip file, manual as pdf files): EUR 0.

Conditions of payment: bank transfer in advance with 2% discount, or by credit card (Master, Visa) net.

#### Key Code

After installation, software has to be released by key code. Key codes will be sent after receipt of payment.

#### HEXAGON Industriesoftware GmbH

Stiegelstrasse 8 D-73230 Kirchheim Tel.+49 702159578 Fax +49 7021 59986  
 Kieler Strasse 1A D-10115 Berlin Mühlstr. 13 D-73272 Neidlingen  
 Mobile: +49 163 7342509 E-Mail: info@hexagon.de Web: http://www.hexagon.de